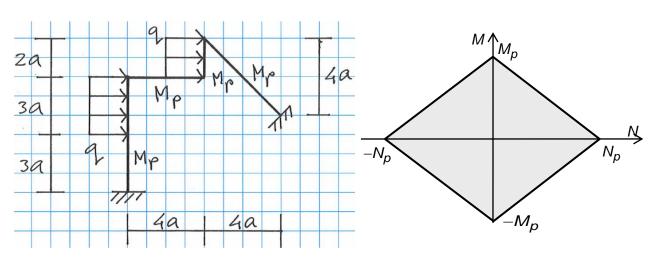
### **Delft University of Technology**

Faculty of Civil Engineering and Geosciences Structural Mechanics Section

### **Exam CIE4150 Plastic Analysis of Structures** Thursday 22 January 2020, 9:00 – 12:00 hours

Write your <u>name</u> and <u>study number</u> at the top of your work.

Also write whether you were a <u>member</u> of the elastic team, plastic team or no team.



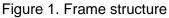


Figure 2. Yield contour

### Problem 1

A frame consists of four members (Fig.1). All members have a strength  $M_p$ . The members are rigidly connected. The supports are fixed. The structure is loaded by two evenly distributed line loads q. The relation of Figure 2 exists between the plastic moments and the plastic normal forces.

$$N_p = \beta \frac{M_p}{a}$$

The influence of shear on the yield contour is neglected. Buckling and second order effects are not considered.

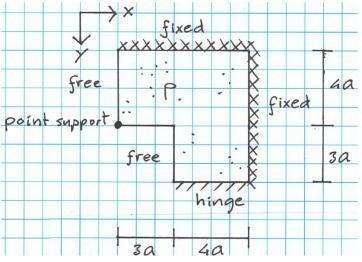
- **a** Assume  $\beta \rightarrow \infty$ . Determine the collapse load *q* for all possible mechanisms. Write the collapse loads as functions of  $M_p$  and *a*. What is the decisive collapse load? (1.5 point)
- **b** Assume  $\beta \rightarrow \infty$ . Draw the bending moment diagram and normal force diagram for the structure at the moment of collapse (1.5 points).
- **c** Assume  $\beta$  = 18. Choose one of the following problems (You need not do both). Determine the largest <u>lower-bound</u> for *q*.

- Determine the smallest <u>upper-bound</u> for *q*.

You only need to write down the equations and not solve the equations (1.5 points).

### Problem 2

A reinforced concrete plate has fixed, hinged and free edges (Fig. 3). It carries an evenly distributed load p [ kN/m<sup>2</sup>]. There is no other load on the plate. The plate is homogeneous and orthotropic.



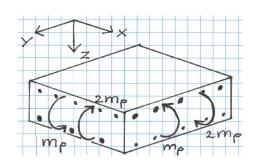


Figure 3. Plate dimensions and reinforcement

**a** Consider the yield line patterns of Figure 4. Which of these patterns give kinematically possible mechanisms? (1 point)

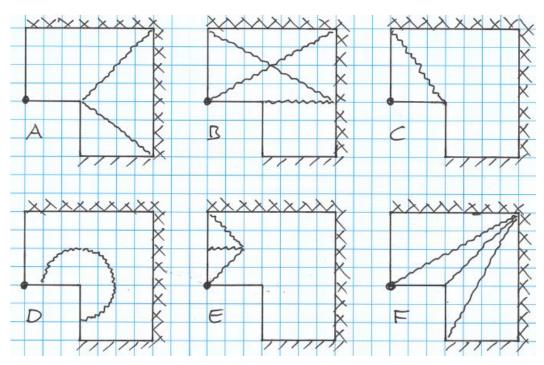


Figure 4. Yield line patterns of problem 2a

**b** Consider the yield line pattern of Figure 5. Determine an <u>upper bound</u> for *p* expressed in  $m_p$  and *a* (1.5 point).

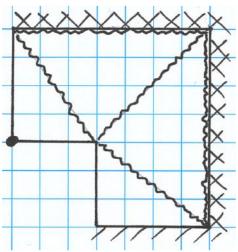


Figure 5. Mechanism of problem 2b

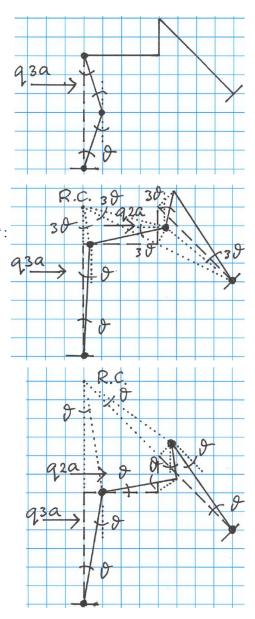
**c** Determine the largest <u>lower-bound</u> for *p* using torsion free beams ( $m_{xy} = 0$ ). You only need to write down the equations and not solve the equations. (1.5 point)

### Problem 3

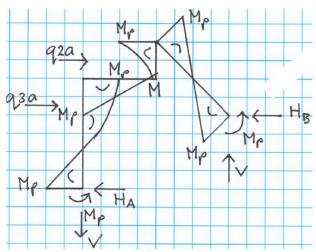
- a Which of the following words does not belong in this list? Choose A, B, C or D (0.5 point).
  - A Yield contour
  - B Limit state function
  - C Response surface
  - D Moment-curvature diagram
- **b** How do we know that the limit load of a plate is exact? Choose A, B, C, or D (0.5 point).
  - A It agrees with the result of a non-linear finite element analysis,
  - B It is the largest lower bound,
  - C It is the largest upper bound,
  - D The upper bound and the lower bound are the same.
- **c** Consider a structure. The degree of indeterminancy is *n*. A mechanism has less than *n*+1 plastic hinges. How do we call this mechanism? (0.5 point)

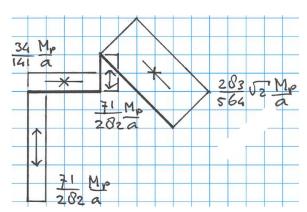
## Answer to problem 1a

> restart:  
> 
$$E := Mp \cdot t + Mp \cdot (t + t) + Mp \cdot t$$
:  
>  $A := q \cdot 3 \cdot a \cdot \frac{3}{2} \cdot a \cdot t$ :  
>  $q := solve(E = A, q); evalf(q);$   
 $q := \frac{8}{9} \frac{Mp}{a^2}$   
 $\frac{0.8888888889 Mp}{a^2}$   
> restart:  
>  $E := Mp \cdot t + Mp \cdot (t + 3 \cdot t) + Mp \cdot (3 \cdot t + 3 \cdot t) + Mp \cdot 3 \cdot t$   
>  $A := q \cdot 3 a \cdot \frac{9}{2} \cdot a \cdot t + q \cdot 2 \cdot a \cdot 3 \cdot a \cdot 3 \cdot t$ :  
>  $q := solve(E = A, q); evalf(q);$   
 $q := \frac{4 Mp}{9 a^2}$   
 $\frac{0.4444444444 Mp}{a^2}$   
> restart:  
>  $E := Mp \cdot t + Mp \cdot (t + t) + Mp \cdot (t + t) + Mp \cdot t$ :  
>  $A := q \cdot 3 \cdot a \cdot \frac{9}{2} a \cdot t + q \cdot 2 \cdot a \cdot 5 \cdot a \cdot t$ :  
>  $q := solve(E = A, q); evalf(q);$   
 $q := \frac{12}{47} \frac{Mp}{a^2}$   
decisive  
 $\frac{0.2553191489 Mp}{a^2}$ 



# Answer to problem 1b

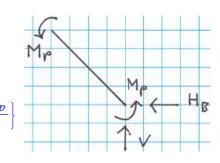


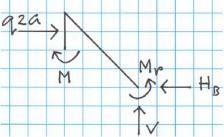


$$\begin{array}{l} > q := \frac{12}{47} \frac{Mp}{a^2} : \\ > eql := q \cdot 3 \cdot a + q \cdot 2 \cdot a = HA + HB : \\ > eq2 := Mp = q \cdot 3 \cdot a \cdot \frac{9}{2} \cdot a + q \cdot 2 \cdot a \cdot 7 \cdot a - Mp - V \cdot 8 \cdot a - HB \cdot 4 \cdot a : \\ > eq3 := Mp = HB \cdot 4 \cdot a - V \cdot 4 \cdot a - Mp : \\ > opl := solve( \{eq1, eq2, eq3\}, \{HA, HB, V\} ); \quad assign(opl) : \\ opl := \left[ HA = \frac{74}{141} \frac{Mp}{a}, HB = \frac{106}{141} \frac{Mp}{a}, V = \frac{71}{282} \frac{Mp}{a} \right] \\ > M := Mp + V \cdot 4 \cdot a - HB \cdot 2 \cdot a - q \cdot 2 \cdot a \cdot a ] \\ M := -\frac{1}{141} Mp \\ -0.007092198582 Mp \\ > N1 := V : N2 := q \cdot 3 \cdot a - HA; N3 := V : N4 := \frac{V}{sqrt(2)} + \frac{HB}{sqrt(2)}; \\ N2 := \frac{34}{141} \frac{Mp}{a} \\ N4 := \frac{283}{564} \frac{Mp\sqrt{2}}{a} \end{array}$$

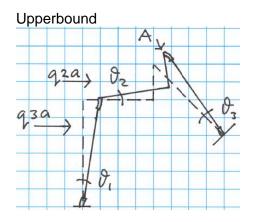
564

а



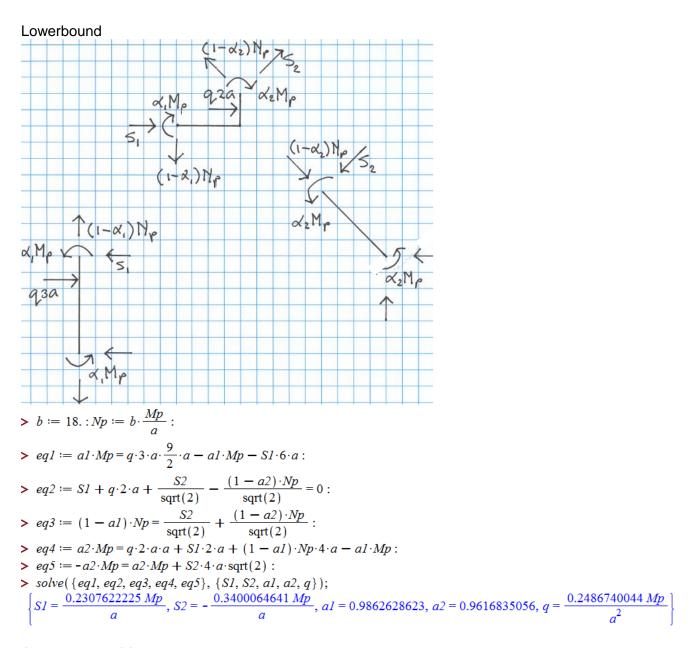


# Answer to problem 1c



> *b* := 18. :

$$= t^{2} \cdot e^{-1} t$$

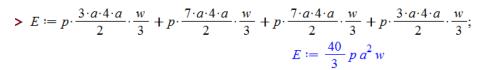


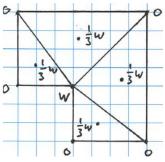
# Answer to problem 2a

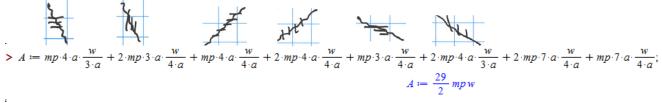
E, F

3 or less correct	0 point
4 correct	0.3 point
5 correct	0.7 point
6 correct	1.0 point

### Answer to problem 2b

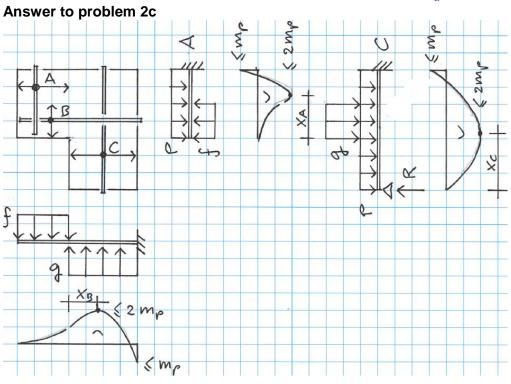






> p := solve(E = A, p); evalf(p);





We cannot use the point support because torsion is neglected.

 $> eq1 := f \cdot 3 \cdot a = g \cdot xB :$   $> eq2 := f \cdot 3 \cdot a \cdot \left(\frac{3}{2} \cdot a + xB\right) - g \cdot xB \cdot \frac{xB}{2} = 2 \cdot mp :$   $> eq3 := g \cdot 4 \cdot a \cdot 2 \cdot a - f \cdot 3 \cdot a \cdot \frac{11}{2} \cdot a = mp :$   $> eq4 := f \cdot 2 = p \cdot xA :$   $> eq5 := f \cdot 2 \cdot a \cdot (xA - a) - p \cdot xA \cdot \frac{xA}{2} = 2 \cdot mp :$   $> eq6 := p \cdot 4 \cdot a \cdot 2 \cdot a - f \cdot 2 \cdot a \cdot 3 \cdot a = mp :$   $> eq6 := p \cdot 4 \cdot a \cdot 2 \cdot a - f \cdot 2 \cdot a \cdot 3 \cdot a = mp :$   $> eq7 := R = p \cdot xC + g \cdot (xC - 3 \cdot a) :$   $> eq8 := R \cdot xC - p \cdot xC \cdot \frac{xC}{2} - g \cdot (xC - 3 \cdot a) \cdot \frac{(xC - 3 \cdot a)}{2} = 2 \cdot mp :$   $> eq9 := p \cdot 7 \cdot a \cdot \frac{7}{2} \cdot a + g \cdot 2 \cdot a \cdot 3 \cdot a - R \cdot 7 \cdot a = mp :$ 

 $\textbf{>} opl := solve(\{eq1, eq2, eq4, eq6, eq7, eq8, eq9\}, \{xA, xB, xC, R, f, g, p\}): assign(opl);$ 

> evalf(p);

> *evalf*(*eq3*);

 $\frac{0.2956414615 mp}{a^2}$ 

-1.844997579 mp = mp > evalf(eq5);

-0.1048478393 mp = 2. mp

## Answer to problem 3

- **a** D
- **b** D
- c Partial mechanism